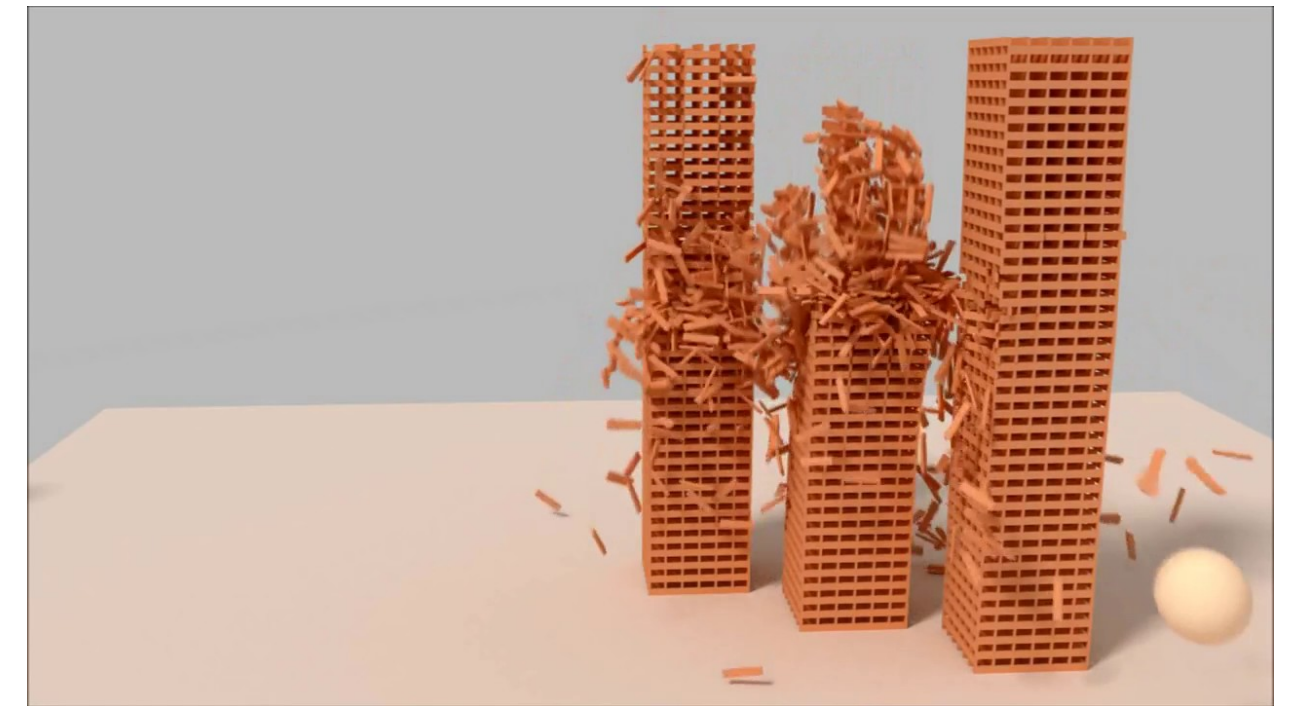
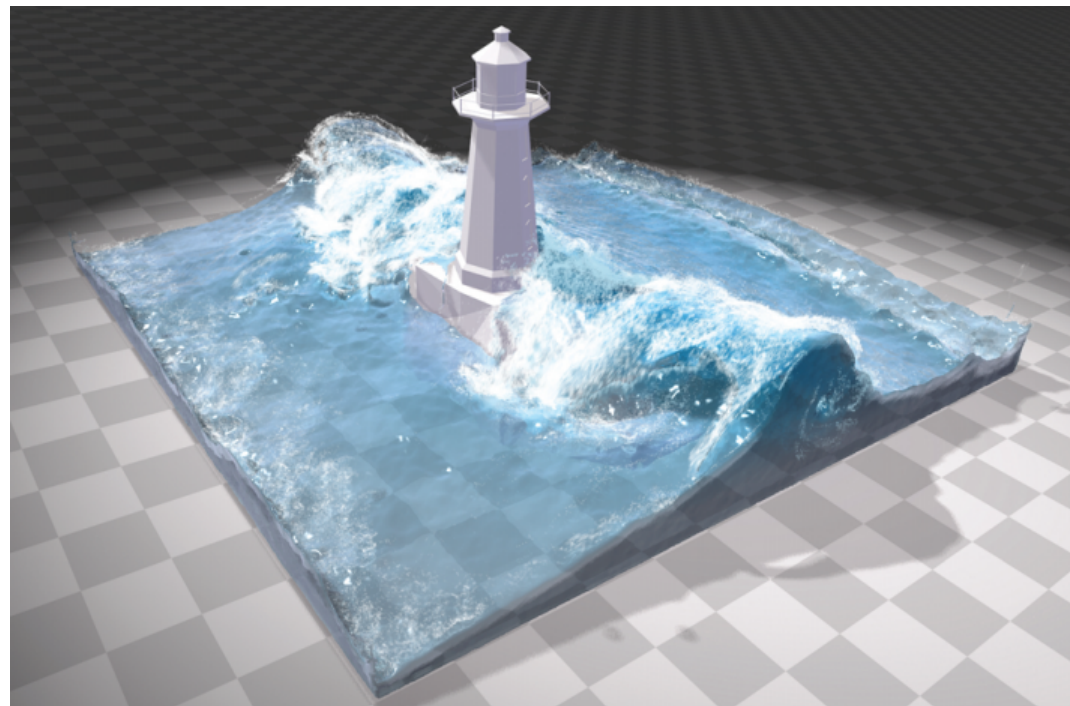


Physically-based simulation

Deformable models

When physically based simulation is needed

- Accurate dynamics
- Tedious to model by hand or procedurally
 - Multiple interacting elements: ex. Multiple collisions: rigid bodies, hairs, etc.
 - Complex animated geometry: Cloths, fluids



Material model

Elasticity: Shape goes back toward its original rest position when external forces are removed.

- Purely elastic models don't lose energy when deformed (potential $\leftrightarrow$ kinetic)



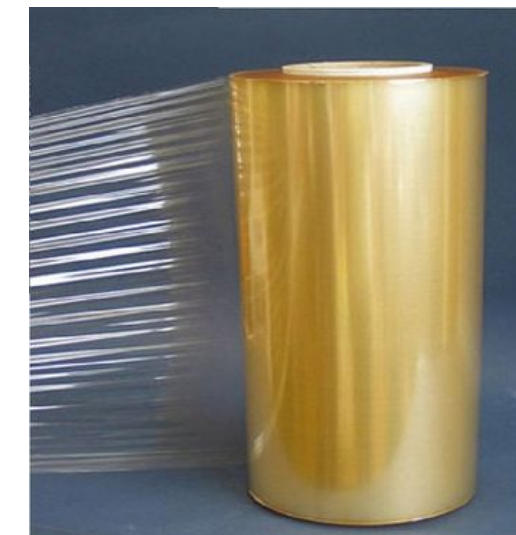
Plasticity: Opposite of elasticity. Plastic material doesn't come back to their original shape (/change their rest position during deformation).

- Ductile material - can allow large amount of plastic deformation without breaking (plastic)
- Brittle - Opposite (glass, ceramics)

Viscosity: Resistance to flow (usually for fluid, ex. honey)

In reality

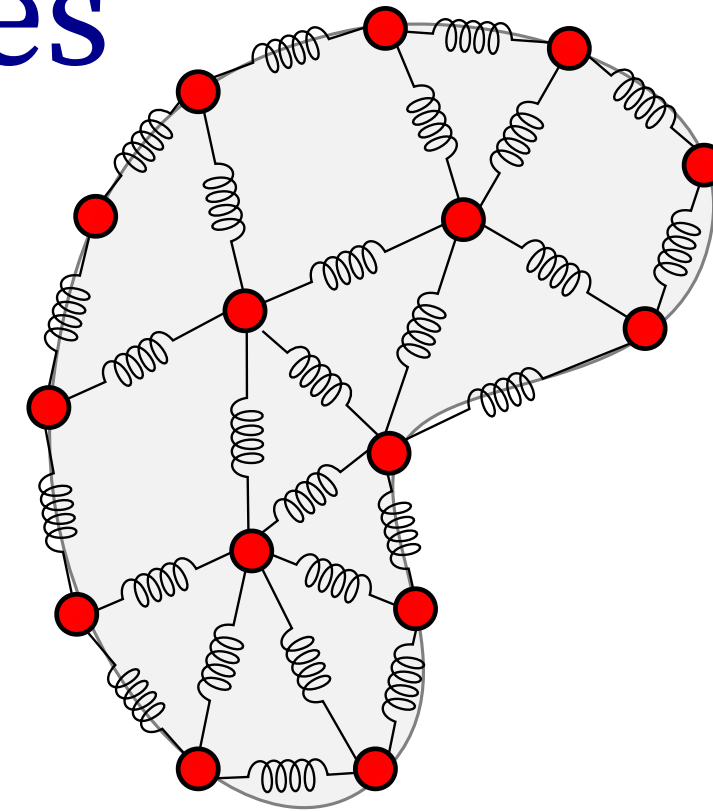
- *Elasto-plastic materials:* Allow elastic behavior for small deformation, and plastic at larger one.
- *Visco-elastic materials:* Elastic properties with delay.



Modeling elastic shapes with particles

Spring mass systems

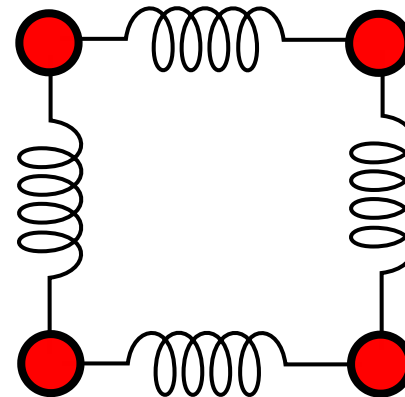
- Particles (position, speed, mass): samples on shape
- Springs : link closed-by particles in the reference shape



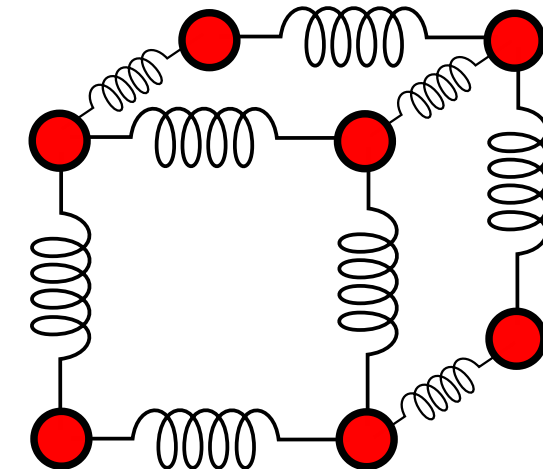
1D curve structure



2D surface structure



3D volume structure

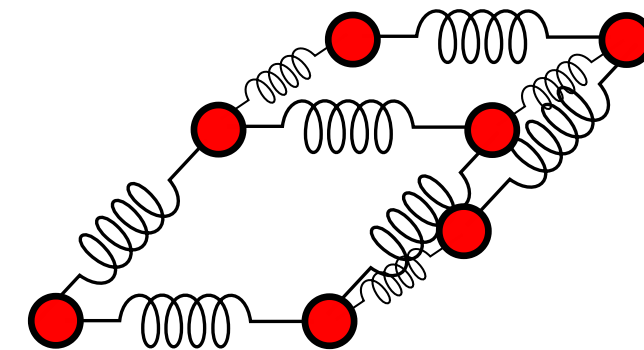
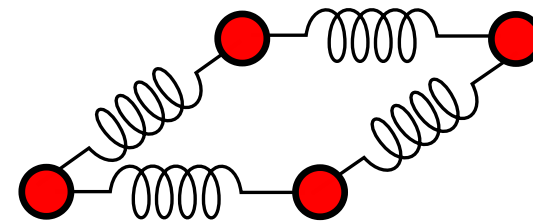
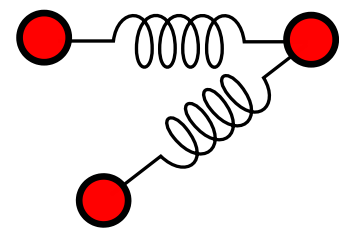


Spring structure

How to model spring connectivity ?

- **Structural springs:** 1-ring neighbors springs ($\simeq$ mesh edges)

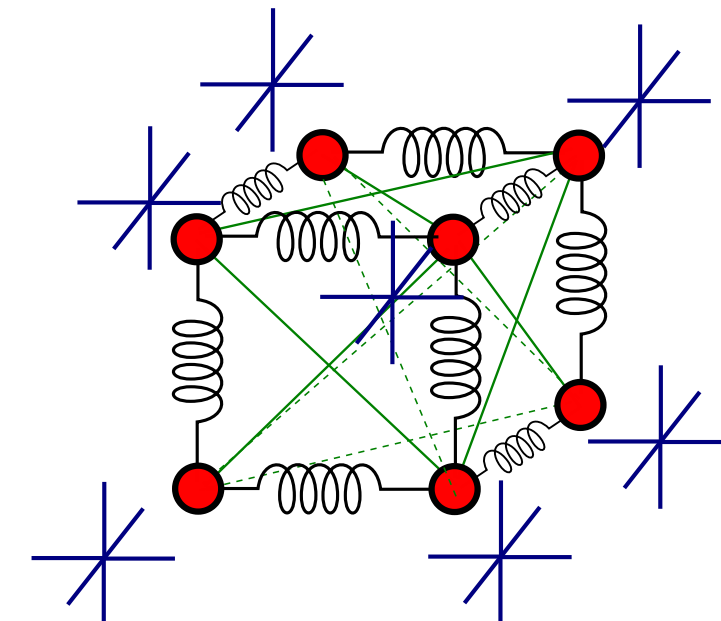
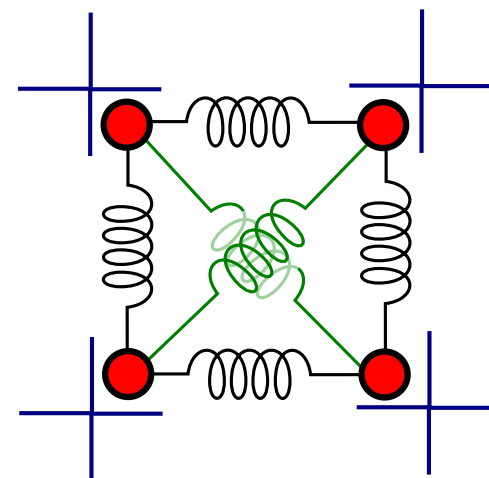
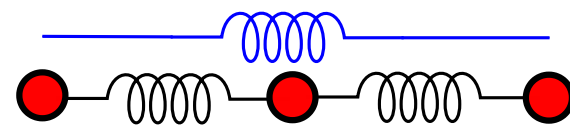
(+) Limit elongation/contraction, (-) Allows shearing, and bending



$\Rightarrow$ Add extra springs connectivity

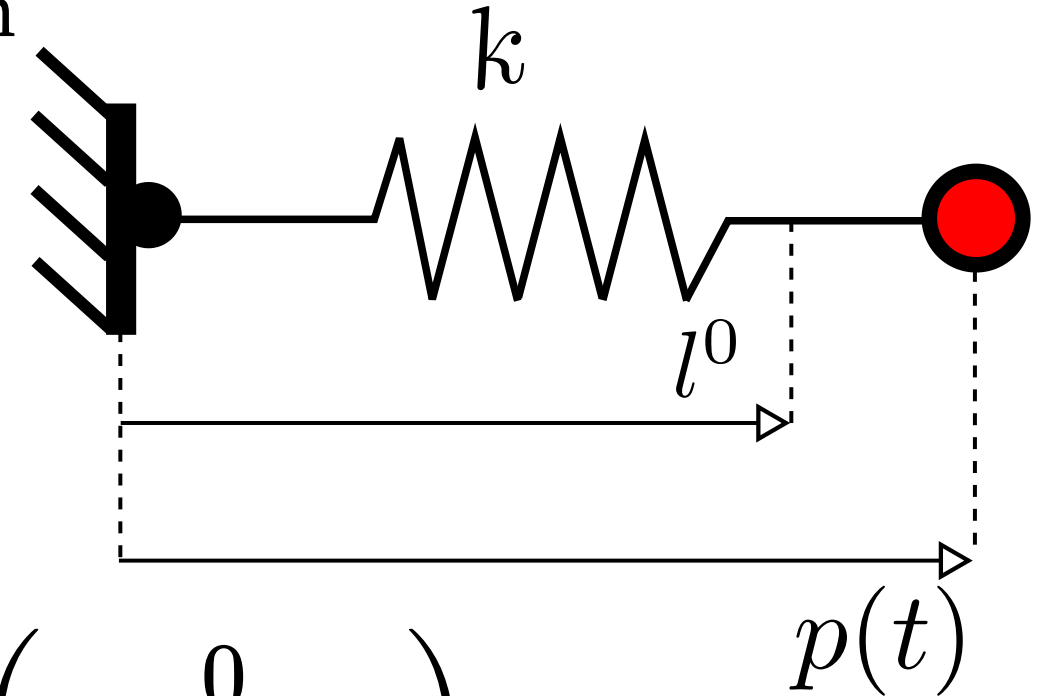
- **Shearing springs:** Diagonal links

- **Bending springs:** 2-ring neighborhood



Example: 1D spring (/Harmonic oscillator)

- Force $F(t) = -k (p(t) - l^0)$, k spring stiffness, l^0 rest length
- Second order differential equation: $m p''(t) + k p(t) = k l^0$



- First order system $\underbrace{\begin{pmatrix} p \\ v \end{pmatrix}}_{u'(t)}'(t) = \underbrace{\begin{pmatrix} v(t) \\ -k/m (p(t) - l^0) \end{pmatrix}}_{\mathcal{F}(u,t)}$

- Linear system $\underbrace{\begin{pmatrix} p \\ v \end{pmatrix}}_{u'(t)}'(t) = \underbrace{\begin{pmatrix} 0 & 1 \\ -k/m & 0 \end{pmatrix}}_A \underbrace{\begin{pmatrix} p \\ v \end{pmatrix}}_{u(t)}(t) + \underbrace{\begin{pmatrix} 0 \\ k/m l^0 \end{pmatrix}}_b$

- Exact solution known: $p(t) = A \sin(\omega t + \varphi) + l^0$

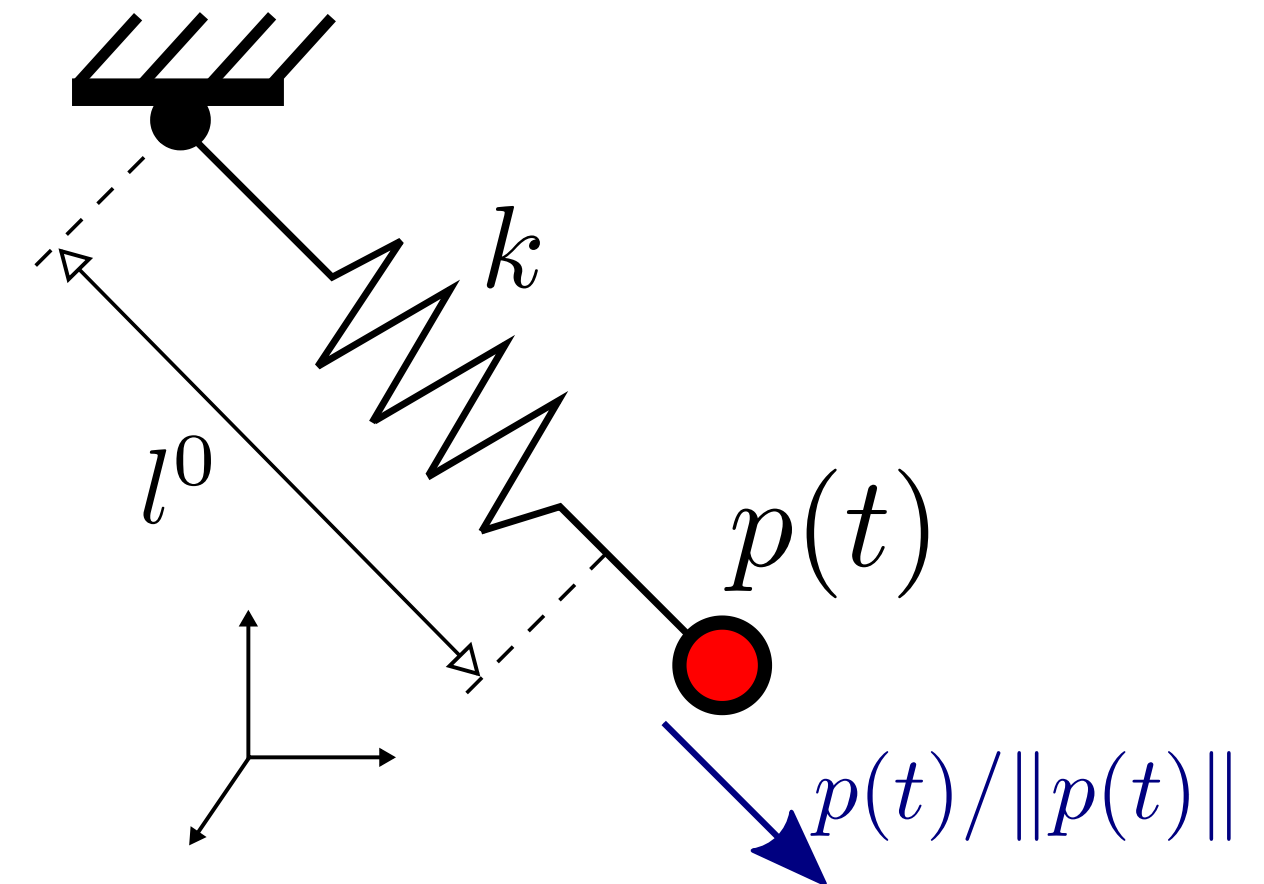
$$\omega = \sqrt{k/m}, A^2 = (p^0 - l^0)^2 + \left(\frac{v_0}{\omega}\right)^2, \tan(\varphi) = \frac{p^0 - l^0}{v^0/\omega}$$

Example: 3D mass spring

- Force $F(t) = m g - k (\|p(t)\| - l^0) \frac{p(t)}{\|p(t)\|}$, k spring stiffness, l^0 rest length
- Second order differential equation: $m p''(t) = m g - k (\|p(t)\| - l^0) \frac{p(t)}{\|p(t)\|}$

- First order system $\underbrace{\begin{pmatrix} p \\ v \end{pmatrix}}_{u'(t)}'(t) = \underbrace{\begin{pmatrix} v(t) \\ g - k/m (\|p(t)\| - l^0) \frac{p(t)}{\|p(t)\|} \end{pmatrix}}_{\mathcal{F}(u,t)}$

- Not linear
- No simple explicit solution



Numerical integration of ODE

General formulation: $u'(t) = \mathcal{F}(u, t)$, $u(t) = (p(t), v(t))$.

Explicit Euler

$$u^{k+1} = u^k + \Delta t \mathcal{F}(u^k, t^k)$$

(+) Easy to implement

(-) Worst scheme in all cases (divergence, low accuracy)

Explicit Runge-Kutta

$$u^{k+1} = u^k + \Delta t \sum_j \alpha_j k_j$$

(+) Good **accuracy**

(+) Efficient to apply

(+/-) Stability OK for non-stiff problem, diverge on stiff problem

(-) Artificial damping for constant energy system

Implicit methods

$$u^{k+1} = u^k + \Delta t \mathcal{F}(u^{k+1}, t^{k+1})$$

(+) Good to deal with **stiff problem** - very stable

(-) Add numerical damping (converge even if solution oscillates)

(-) Hard/computationally costly to apply on non linear problem

Symplectic integrator

$$v^{k+1} = v^k + \Delta t F^k / m$$

$$p^{k+1} = p^k + \Delta t v^{k+1}$$

(+) Handle well constant energy system, preserves energy (Hamiltonian systems)

(+) Simple and efficient to implement

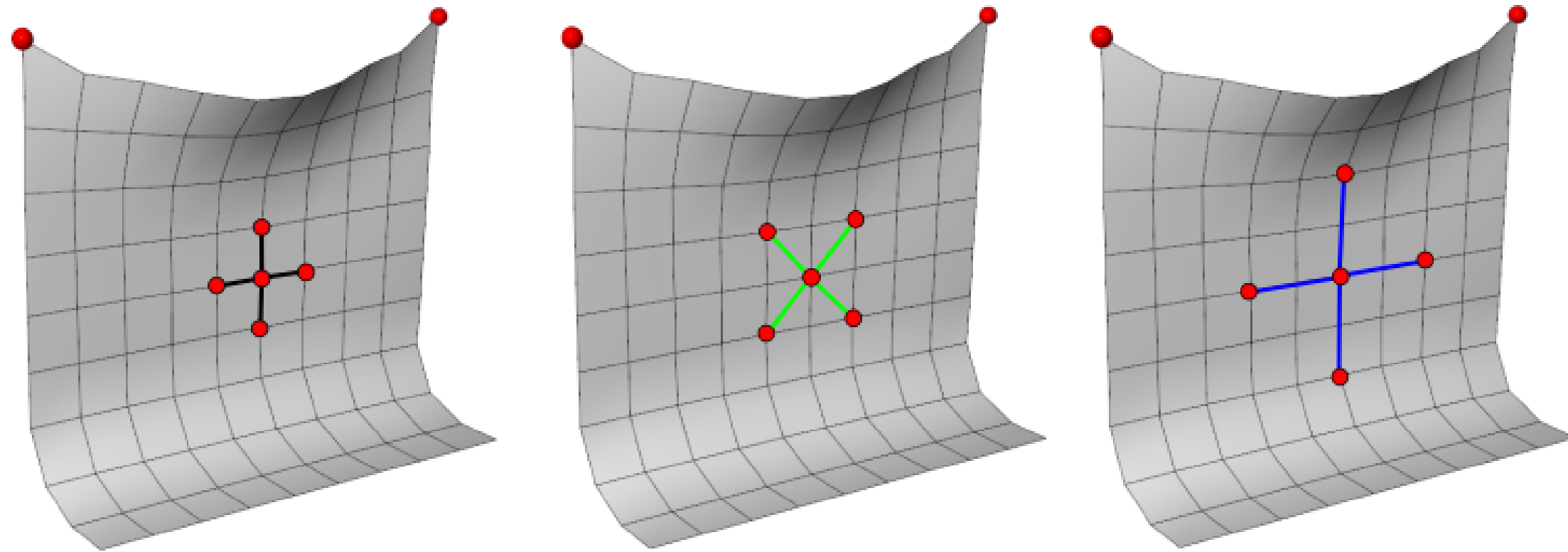
(-) Less accurate than RK

(-) Diverge on stiff problem

Cloth Simulation

Mass-spring cloth simulation

- Particles are sampled on a $N \times N$ grid.
 - Each particle has a mass m ($m_{cloth} = N^2 m$)
- Set structural, shearing and bending springs.



Forces

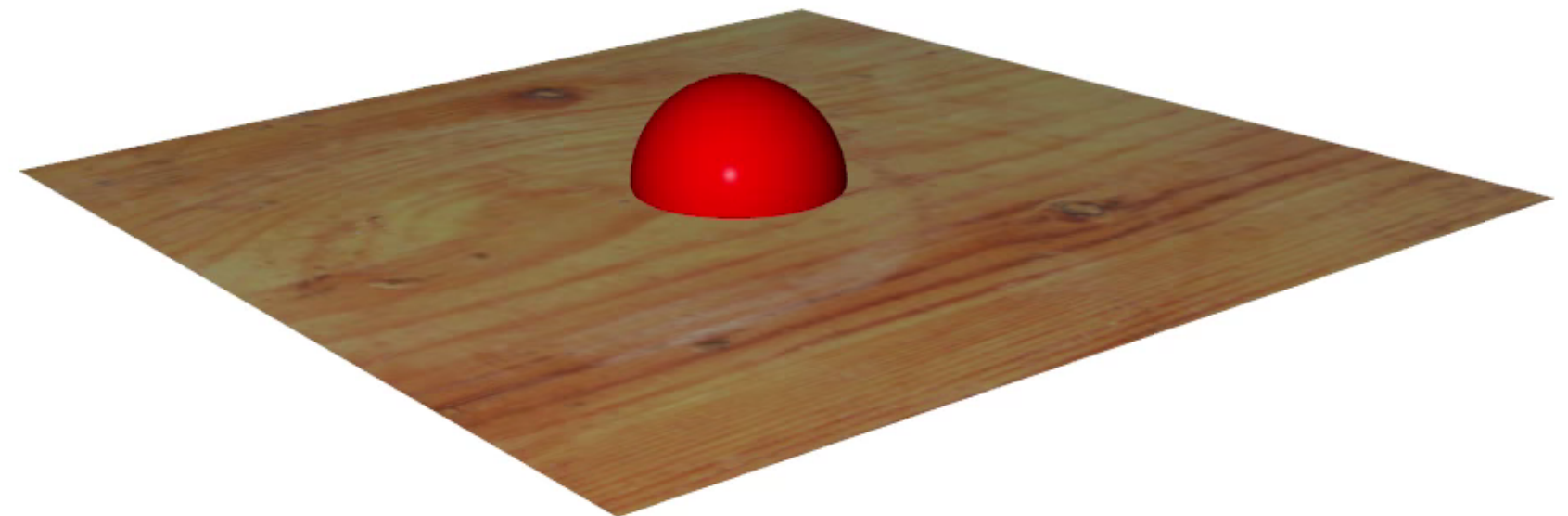
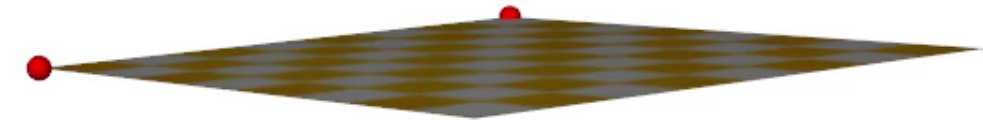
- On each particle: gravity + drag + spring forces

$$- F_i(p, v, t) = m_i g - \mu v_i(t) + \sum_{j \in \mathcal{V}_i} K_{ij} \left(\|p_j(t) - p_i(t)\| - L_{ij}^0 \right) \frac{p_j(t) - p_i(t)}{\|p_j(t) - p_i(t)\|}$$

- $\mathcal{V}_i$: neighborhood of particle i

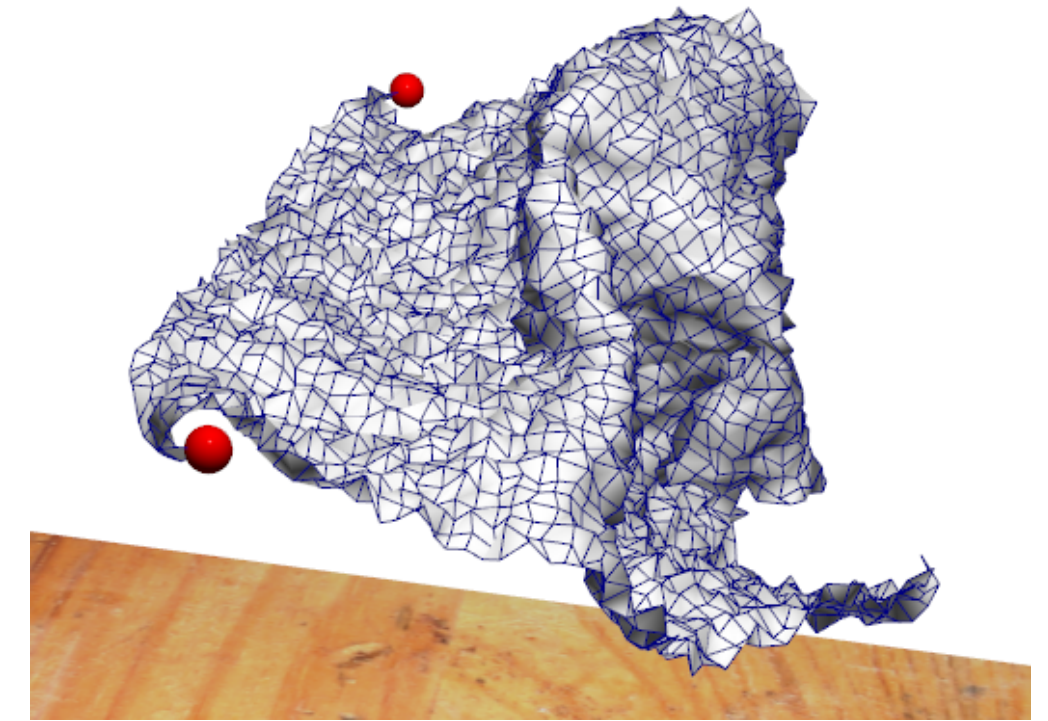
- L_{ij}^0 : rest length of spring ij

Associated ODE $\forall i, \begin{cases} p_i'(t) = v_i(t) \\ v_i'(t) = F_i(p, v, t) \end{cases}$



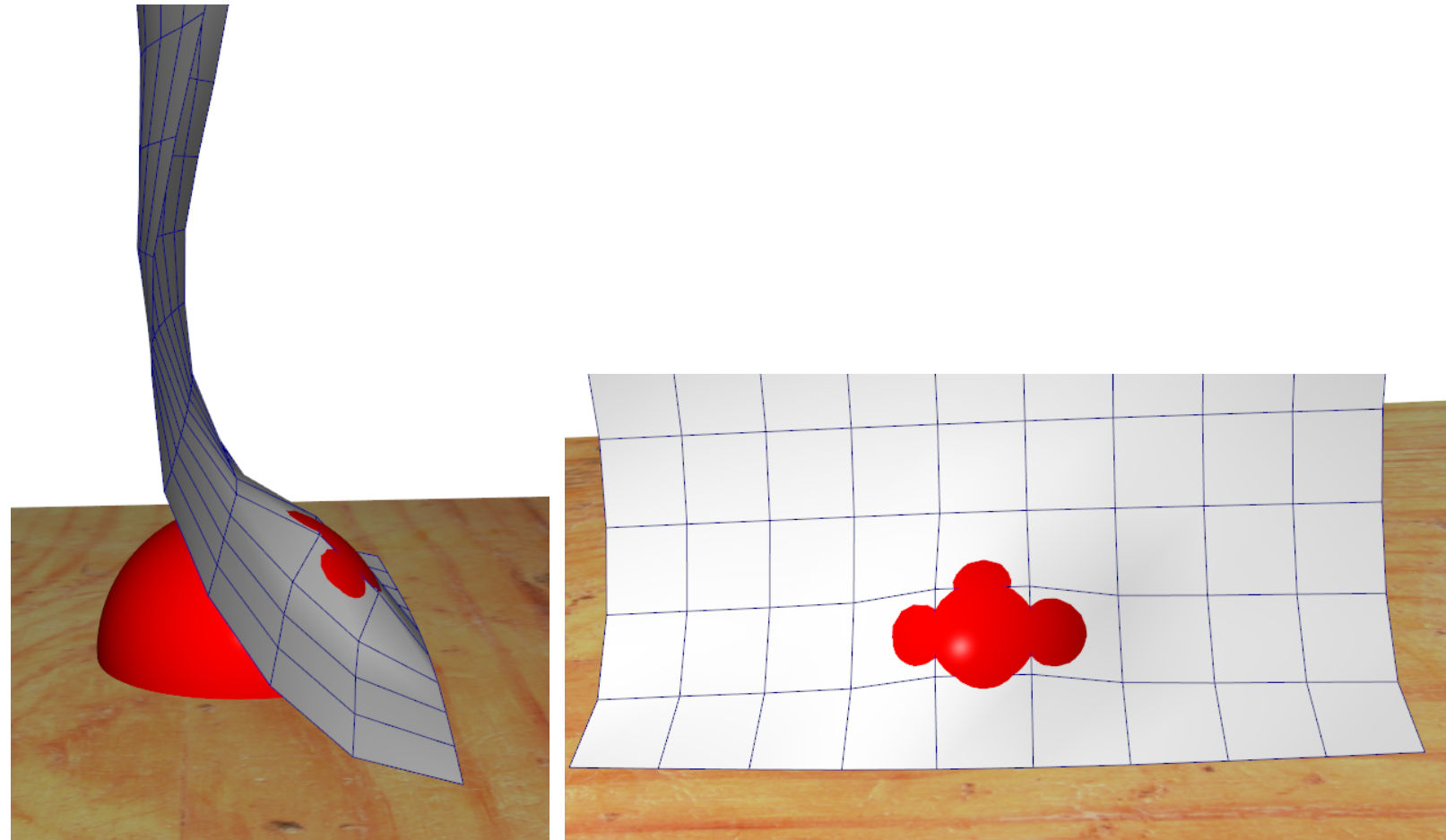
Note on Mass-Spring numerical solution

- Non-linear ODE
- Large K_{ij} : good length preservation, but stiff ODE
⇒ divergence of explicit schemes.
- Avoid explicit Euler (divergence)
- Semi-implicit Euler/Verlet works fine for low K_{ij}
Semi-implicit Euler + PBD allows simple integration + stable stiff springs
[Muller et al. [PBD](#) , [Inextensible clothing in Computer Games](#)]
- RK4 more accurate (but higher complexity than Verlet)
- Implicit Euler : requires linearization, but very stable



Collisions

- Simple approach : Handled as collision between particles and shapes
 - (+) Simple and efficient
 - (-) Collision may still appears within a triangle
- ⇒ Exhaustive approach: point-face + edge-edge



Limitation of mass spring model and continuous model

- Does mass-spring system converge toward a unique solution when sampling increase ?

⇒ No :(

Depends on the connectivity → bad for physical accuracy

Corollary

- Mass-springs work well for grid-mesh structure (draping)
- Less for arbitrary triangular meshes

1st improvment: Change toward energy formulation for bending springs (limits locking effect)

$$F = \frac{\partial E}{\partial p}$$

$$E = \frac{1}{2} K L \kappa^2, \kappa: \text{curvature}$$

[Cho et al, Stable but Responsive Cloth, ACM SIGGRAPH 2002]



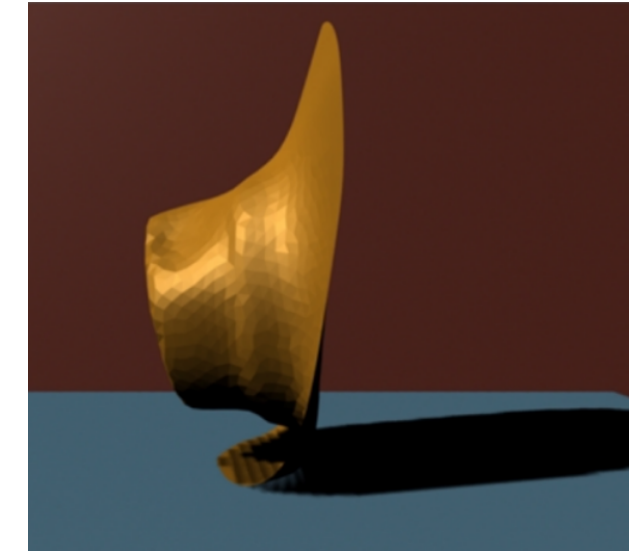
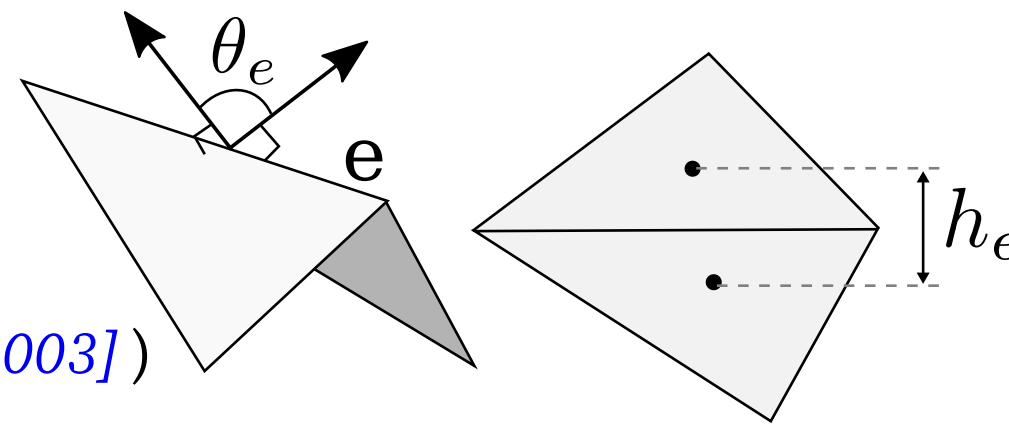
Triangle as continuous elements

- Defining Bending Energy between triangles

$$- W_B(x) = \sum_{\text{edges } e} (\theta_e - \theta_e^0) \frac{\|e^0\|}{h_e^0}$$

[E. Grinspun et al., Discrete Shells, SCA 2003]

(or expressed using forces in [R. Bridson et al., SCA 2003])



- Going toward full FEM numerical resolution

- B. Thomaszewski et al. [SCA 2006], [VRIPHYS 2008], [EG 2009].

